

Chapter 3 - Solved Problems

Solved Problem 3.1. A nonlinear system has an input-output model given by

$$\frac{dy(t)}{dt} + (1 + 0.2y(t))y(t) = u(t) + 0.2u(t)^3 \quad (1)$$

3.1.1 Compute the operating point(s) for $u_Q = 2$. (assume it is an equilibrium point)

3.1.2 Obtain a linearized model for each of the operating points above.

Solutions to Solved Problem 3.1

Solved Problem 3.2. A nonlinear system is described in state space form by the model

$$\dot{x}_1(t) = -x_1(t)^2 + x_2(t) + 3u(t) \quad (2)$$

$$\dot{x}_2(t) = -2x_1(t)x_2(t) \quad (3)$$

$$y(t) = x_1(t) \quad (4)$$

Obtain a linearized model around the equilibrium point $(u_Q, y_Q) = (2, 0)$.

Solutions to Solved Problem 3.2

Solved Problem 3.3. Consider a discrete time system with input $u[k]$ and output $y[k]$, having an input-output model given by

$$y[k] + 0.4y[k - 1] = u[k - 2] \quad (5)$$

Choose state variables and build a state space model

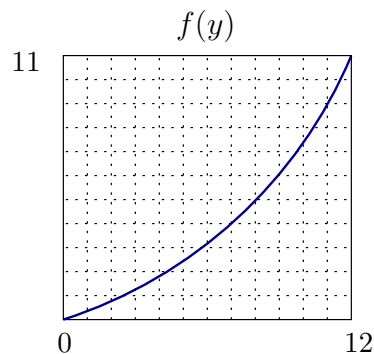
Solutions to Solved Problem 3.3

Solved Problem 3.4. The input-output model for a nonlinear system is given by

$$\frac{dy(t)}{dt} + f(y) = 2u(t) \quad (6)$$

where $f(y)$ is the nonlinear function appearing in the figure.

Build a linearized model for the equilibrium point determined by $u_Q = 3$.



Solutions to Solved Problem 3.4

Solved Problem 3.5. Consider the electric network shown in Figure 1

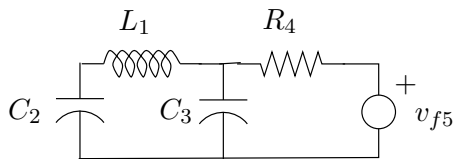


Figure 1: Electric network

3.5.1 Without using any equations, discuss how many states the system has.

3.5.2 Build a state space model.

Solutions to Solved Problem 3.5

Solved Problem 3.6. Consider a single tank of constant cross-sectional area A . The flow of water from the tank is governed by the relationship

$$f_{out} = K\sqrt{h} \tag{7}$$

where h is the height of liquid in the tank and K is a constant.

Assume that the flow of liquid into the tank is a control variable, u .

3.6.1 Write down the equation governing the height of liquid in the tank.

3.6.2 Linearize the model about a nominal height of $h = h^*$.

3.6.3 Repeat part (i) and (ii) for a tank where the cross sectional area increases with height i.e., $A = ch$.

Solutions to Solved Problem 3.6

Solved Problem 3.7. Consider a ball in a frictionless cone which is being rotated as shown in Figure 2. Write down the equations of motion of the ball in the vertical plane.

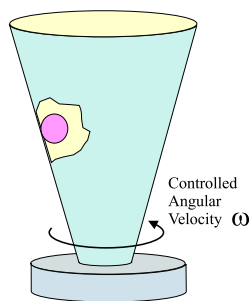


Figure 2: Cone

Solutions to Solved Problem 3.7

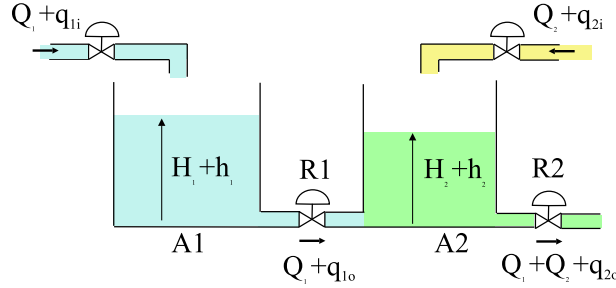


Figure 3: Two Tanks

Solved Problem 3.8. *Contributed by - James Welsh, University of Newcastle, Australia.*

Consider the two tanks system shown in Figure 3:

- Q_1 & Q_2 are steady state flows
- H_1 & H_2 are steady state heights (head)
- R_1 & R_2 are valve resistances

All lower case variables are considered to be small quantities.

Find a state space model for the system using h_1 and h_2 as the state variables and with q_{1i} and q_{2i} as the inputs.

Solutions to Solved Problem 3.8

Solved Problem 3.9. *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

Build a linear model around the equilibrium point defined by $u_Q = \sqrt{6}$ for the system:

$$\frac{d^2 y(t)}{dt^2} + y(t) \frac{dy(t)}{dt} + y^3(t) - y(t) = 2 \frac{du(t)}{dt} + u^2(t) \quad (8)$$

Solutions to Solved Problem 3.9

Solved Problem 3.10. *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

Build a state space model for the system with input $u(t)$ and output $y(t)$ and having a model given by the differential equation:

$$\frac{d^2 y(t)}{dt^2} + 3 \frac{dy(t)}{dt} + y(t) = 2u(t) \quad (9)$$

Solutions to Solved Problem 3.10

Solved Problem 3.11. *Contributed by - Alvaro Liendo, Universidad Tecnica Federico Santa Maria, Chile.*

Build a state space model for the system with input $u(t)$ and output $y(t)$ and having a model given by the differential equation

$$\frac{d^2 y(t)}{dt^2} + 3 \frac{dy(t)}{dt} + y(t) = 2 \frac{du(t)}{dt} \quad (10)$$

Chapter 3 - Solutions to Solved Problems

Solution 3.1.

3.1.1 The operating point (u_Q, y_Q) must satisfy

$$(1 + 0.2y_Q)y_Q = u(t) + 0.2u_Q^3 \implies 0.2y_Q^2 + y_Q - 3.6 = 0 \quad (11)$$

This yields two operating points, P_1 and P_2 given by $(2, -7.4244)$ and $(2, 2.4244)$ respectively.

3.1.2 To obtain the linearized models we can proceed in many ways. For instance, we can apply the method outlined in section §3.10. To do that we define the state as $x(t) = y(t)$. We thus have $x_Q = y_Q$ and

$$\dot{x}(t) = f(x(t), u(t)) = -x(t) - 0.2x(t)^2 + u(t) + 0.2u(t)^3 \quad (12)$$

$$y(t) = g(x(t), u(t)) = x(t) \quad (13)$$

If we define $x(t) = x_Q + \Delta x(t)$, $u(t) = u_Q + \Delta u(t)$, then

$$\frac{d\Delta x(t)}{dt} = \left. \frac{\partial f}{\partial x} \right|_{\substack{x=x_Q \\ u=u_Q}} \Delta x(t) + \left. \frac{\partial f}{\partial u} \right|_{\substack{x=x_Q \\ u=u_Q}} \Delta u(t) = (-1 - 0.4x_Q)\Delta x(t) + (1 + 0.6u_Q)\Delta u(t) \quad (14)$$

$$\Delta y(t) = \Delta x(t) \quad (15)$$

We can also express this in input-output form as

$$\frac{d\Delta y(t)}{dt} + (1 + 0.4y_Q)\Delta y(t) = (1 + 0.6u_Q)\Delta u(t) \quad (16)$$

For the two operating points described above, we have

$$P_1 : \frac{d\Delta y(t)}{dt} - 1.9698\Delta y(t) = 2.2\Delta u(t) \quad (17)$$

$$P_2 : \frac{d\Delta y(t)}{dt} + 1.9698\Delta y(t) = 2.2\Delta u(t) \quad (18)$$

Solution 3.2. We first need to compute the state, (x_{1Q}, x_{2Q}) , corresponding to the equilibrium point. We notice that $x_{1Q} = y_Q = 0$, and from the first state equation we have that

$$0 = x_{2Q} + 3u_Q \iff x_{2Q} = -3u_Q = 6 \quad (19)$$

The reader can readily verify that these values also satisfy the second state equation at the equilibrium point.

We next express the state and plant input, output as

$$x_1(t) = x_{1Q} + \Delta x_1(t); \quad x_2(t) = x_{2Q} + \Delta x_2(t) \quad (20)$$

$$u(t) = u_Q + \Delta u(t); \quad y(t) = y_Q + \Delta y(t) \quad (21)$$

and we finally use the method presented in section §3.10 of the book, leading to

$$\frac{d\Delta x_1(t)}{dt} = -2x_{1Q}\Delta x_1(t) + \Delta x_2(t) + 3\Delta u(t) = \Delta x_2(t) + 3\Delta u(t) \quad (22)$$

$$\frac{d\Delta x_2(t)}{dt} = -2x_{2Q}\Delta x_1(t) - 2x_{1Q}\Delta x_2(t) = -12\Delta x_1(t) \quad (23)$$

$$\Delta y(t) = \Delta x_1(t) \quad (24)$$

Solution 3.3. *The interesting aspect of this problem is the two unit delay on the input.*

The state variables must include all information we require to know at time $k = k_o$ such that, given the input $u[k]$, for all $k \geq k_o$, we are able to compute $y[k]$, for all $k > k_o$. From the system equation we have

$$y[k_o + 1] = u[k_o - 1] - 0.4y[k_o] \quad (25)$$

$$y[k_o + 2] = u[k_o] - 0.4y[k_o + 1] \quad (26)$$

We see that we require to know $y[k_o]$, and $u[k_o - 1]$ to predict the future response. We thus choose

$$x_1[k] = y[k]; \quad x_2[k] = u[k - 1]; \quad (27)$$

and we notice that

$$x_1[k + 1] = y[k + 1] = u[k - 1] - 0.4y[k] = x_2[k] - 0.4x_1[k] \quad (28)$$

$$x_2[k + 1] = u[k] \quad (29)$$

$$y[k] = x_1[k] \quad (30)$$

Setting the above in matrix form, we finally obtain

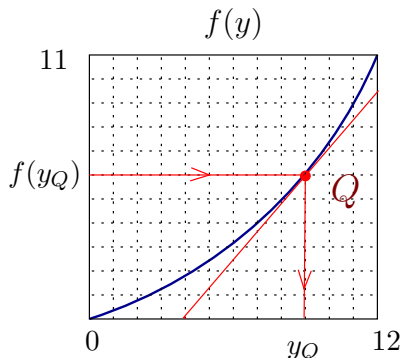
$$\begin{bmatrix} x_1[k + 1] \\ x_2[k + 1] \end{bmatrix} = \begin{bmatrix} -0.4 & 1 \\ 0 & 0 \end{bmatrix} \begin{bmatrix} x_1[k] \\ x_2[k] \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u[k] \quad (31)$$

$$y[k] = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} x_1[k] \\ x_2[k] \end{bmatrix} \quad (32)$$

Solution 3.4. *The linearized model has the form*

$$\frac{d\Delta y(t)}{dt} + \left. \frac{df(y)}{dy} \right|_Q \Delta y(t) = 2\Delta u(t) \quad (33)$$

where $y(t) = y_Q + \Delta y(t)$ and $u(t) = u_Q + \Delta u(t)$. We thus need to compute the derivative of $f(y)$ at the equilibrium point. This is done graphically as shown below



We note that, at an equilibrium point, $f(y_Q) = 2u_Q$. Thus, in our particular example, $f(y_Q) = 6$. With this value we find the operating point. The slope of the tangent at that point is the derivative of $f(y)$ at Q , and is approximately equal to 1.19. Hence the linearized model is given by

$$\frac{d\Delta y(t)}{dt} + 1.19\Delta y(t) = 2\Delta u(t) \quad (34)$$

Solution 3.5.

3.5.1 The number of states required for the system is equal to the number of initial conditions we can *arbitrarily* set in the network. In this case, the number is three: an initial current in inductor L_1 and the initial voltages on C_1 and C_2 . The reader may note that the answer can be different if we allow some singular cases. For instance, if we make $R_4 = 0$, then the number of states required will be only two, since then the voltage in C_3 will always be equal to the source voltage and hence it can not be set *arbitrarily*.

3.5.2 To build the state space model we choose as state variables the electric signals $i_1(t)$, $v_2(t)$ and $v_3(t)$ which are shown in the network schematic shown in Figure 4.

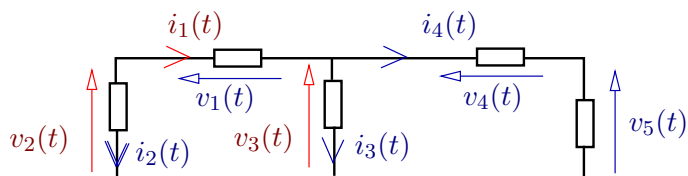


Figure 4: Electric network skeleton

Applying Kirchoff's laws and component laws we obtain

$$v_5(t) = v_{f5}(t) = -v_4(t) + v_3(t) = -R_4 i_4(t) + v_3(t) \quad (35)$$

$$i_4(t) = i_1(t) - i_3(t) = i_1(t) - C_3 \frac{dv_3(t)}{dt} \quad (36)$$

$$v_2(t) = v_1(t) + v_3(t) = L_1 \frac{di_1(t)}{dt} + v_3(t) \quad (37)$$

$$i_1(t) = -i_2(t) = -C_2 \frac{dv_2(t)}{dt} \quad (38)$$

We also notice that the system input is the voltage of the independent voltage source, $v_{f5}(t)$. Rearranging the previous equations we finally obtain

$$\frac{di_1(t)}{dt} = \frac{1}{L_1}v_2(t) - \frac{1}{L_1}v_3(t) \quad (39)$$

$$\frac{dv_2(t)}{dt} = -\frac{1}{C_2}i_1(t) \quad (40)$$

$$\frac{dv_3(t)}{dt} = -\frac{1}{C_3}i_1(t) - \frac{1}{R_4C_3}v_3(t) + \frac{1}{R_4C_3}v_{f5}(t) \quad (41)$$

Solution 3.6.

3.6.1 *The volume of liquid in the tank is*

$$V = Ah \quad (42)$$

The rate of change of liquid in the tank is

$$\frac{dV}{dt} = u - f_{out} \quad (43)$$

$$A\frac{dh}{dt} = u - K\sqrt{h} \quad (44)$$

$$\text{or} \quad \frac{dh}{dt} = -\frac{K}{A}\sqrt{h} - \frac{1}{A}u \quad (45)$$

3.6.2 *If $h = h^*$, then $u = K\sqrt{h^*}$ for a steady height.*

Let $\tilde{h} = h - h^$, $\tilde{u} = u - u^*$.*

Then linearizing (44) we obtain

$$A\frac{d(h^* + \tilde{h})}{dt} = u^* + \tilde{u} - K\sqrt{h^* + \tilde{h}} \quad (46)$$

$$\simeq u^* + \tilde{u} - K\sqrt{h^*} - \frac{K}{2}(h^*)^{\frac{1}{2}}\tilde{h} \quad (47)$$

Hence

$$\frac{d\tilde{h}}{dt} \simeq -\frac{K}{A2\sqrt{h^*}}\tilde{h} - \frac{1}{A}\tilde{u} \quad (48)$$

3.6.3 *Here*

$$V = Ah = ch^2 \quad (49)$$

Hence

$$\frac{dV}{dt} = u - f_{out} \quad (50)$$

$$c \frac{dh^2}{dt} = u - K\sqrt{h} \quad (51)$$

$$2ch \frac{dh}{dt} = u - K\sqrt{h} \quad (52)$$

$$\frac{dh}{dt} = -\frac{K}{2c\sqrt{h}} + \frac{u}{2ch} \quad (53)$$

Linearizing about h^* gives

$$u^* = K\sqrt{h^*} \quad \text{as before} \quad (54)$$

(Note that this is reasonable since we have to balance the outflow by the inflow).

Also

$$\frac{d\tilde{h}}{dt} \simeq -\frac{K}{2c\sqrt{h^*}} + \frac{K}{4c}(h^*)^{-\frac{3}{2}}\tilde{h} - \frac{u^*}{2ch^*} - \frac{\tilde{u}}{2ch^*} - \frac{u^*}{2c}(h^*)^{-2}\tilde{h} \quad (55)$$

$$= \left[\frac{K}{4c}(h^*)^{-\frac{3}{2}} - \frac{u^*}{2c}(h^*)^{-2} \right] \tilde{h} - \frac{\tilde{u}}{2ch^*} \quad (56)$$

Solution 3.7. We assume that the cone makes an angle θ with the horizontal plane. Also, assume that the diameter of the base of the cone is d_o . Also let h denote the height of the ball.

Then resolving the forces on the ball tangential to the wall, we have

$$mg \cos \theta = m \left\{ \cot[h + \frac{d_o}{2} \tan \theta] \right\} \omega^2 \sin \theta \quad (57)$$

Differentiation with respect to time gives

$$-csc[[h + \frac{d_o}{2} \tan \theta]^2 \omega^2 \sin \theta \dot{h} + \cot[h + \frac{d_o}{2} \tan \theta] 2\omega \dot{\omega} \sin \theta = 0 \quad (58)$$

Solution 3.8. Because we are only interested in small variations, we can assume that flow through the valves is linearly related to the difference in head.

The equations for Tank 1 then become:

$$A_1 \frac{dh_1}{dt} = q_{1i} - q_{1o} \quad (59)$$

$$q_{1o} = \frac{h_1 - h_2}{R_1} \quad (60)$$

The corresponding equations for Tank 2 are

$$A_2 \frac{dh_2}{dt} = q_{1o} + q_{2i} - q_{2o} \quad (61)$$

$$q_{2o} = \frac{h_2}{R_2} \quad (62)$$

Substituting 60 into 59

$$\frac{dh_1}{dt} = \frac{1}{A_1} \left(q_{1i} - \frac{h_1 - h_2}{R_1} \right) \quad (63)$$

Substituting 60 and 62 into 61

$$\frac{dh_2}{dt} = \frac{1}{A_2} \left(\frac{h_1 - h_2}{R_1} + q_{2i} - \frac{h_2}{R_2} \right) \quad (64)$$

We now define the state variables as

$$x_1 = h_1 \quad (65)$$

$$x_2 = h_2 \quad (66)$$

We then write 63 and 64 respectively as

$$\dot{x}_1 = -\frac{1}{R_1 A_1} x_1 + \frac{1}{R_1 A_1} x_2 + \frac{1}{A_1} q_{1i} \quad (67)$$

$$\dot{x}_2 = -\frac{1}{R_1 A_2} x_1 - \left(\frac{1}{R_1 A_2} + \frac{1}{R_2 A_2} \right) x_2 + \frac{1}{A_2} q_{2i} \quad (68)$$

In summary, the state space model is

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} -\frac{1}{R_1 A_1} & \frac{1}{R_1 A_1} \\ \frac{1}{R_1 A_2} & -\left(\frac{1}{R_1 A_2} + \frac{1}{R_2 A_2} \right) \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} \frac{1}{A_1} & 0 \\ 0 & \frac{1}{A_2} \end{bmatrix} \begin{bmatrix} q_{1i} \\ q_{2i} \end{bmatrix} \quad (69)$$

$$\begin{bmatrix} h_1 \\ h_2 \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} \quad (70)$$

Solution 3.9. At equilibrium $\frac{d^n y(t)}{dt^n}$ and $\frac{d^n y(t)}{dt^n}$ are zero. Hence at the equilibrium point, the differential equation reduces to

$$y_Q^3 - y_Q = u_Q^2 \quad (71)$$

This leads to $(u_Q, y_Q) = (\sqrt{6}, -2)$. The other 2 solutions of the third degree equation are complex numbers.

The linearized model is obtained using a first order Taylor series approximation , which is as follows:

$$f(x(t), y(t)) \approx f(x_Q, y_Q) + \left. \frac{\partial f}{\partial x} \right|_{\substack{x=x_Q \\ y=y_Q}} \Delta x + \left. \frac{\partial f}{\partial y} \right|_{\substack{x=x_Q \\ y=y_Q}} \Delta y \quad (72)$$

Considering $u(t) = u_Q + \Delta u(t)$ and $y(t) = y_Q + \Delta y(t)$, we obtain

$$y(t) \frac{dy(t)}{dt} \approx y_Q \frac{d\Delta y(t)}{dt} \quad (73)$$

$$y^3(t) \approx 3y_Q^2 \Delta y(t) + y_Q^3 \quad (74)$$

$$u^2(t) \approx 2u_Q \Delta u(t) + u_Q^2 \quad (75)$$

Thus

$$\frac{d^2 \Delta y(t)}{dt^2} + y_Q \frac{d\Delta y(t)}{dt} + 3y_Q^2 \Delta y(t) + y_Q^3 - \Delta y(t) - y_Q = 2 \frac{d\Delta u(t)}{dt} + 2u_Q \Delta u(t) + u_Q^2 \quad (76)$$

Using equation (71) and the value of u_Q and y_Q the linear model is

$$\frac{d^2 \Delta y(t)}{dt^2} - 2 \frac{d\Delta y(t)}{dt} + 12\Delta y(t) - \Delta y(t) = 2 \frac{d\Delta u(t)}{dt} + 2\sqrt{6}\Delta u(t) \quad (77)$$

Solution 3.10. Consider the states $x_1(t) = \frac{dy(t)}{dt}$ and $x_2(t) = y(t)$. We then have the differential equation

$$\dot{x}_1(t) + 3x_1(t) + x_2(t) = 2u(t) \quad (78)$$

Thus a suitable state space model is given by

$$\dot{x}_1(t) = -3x_1(t) - x_2(t) + 2u(t) \quad (79)$$

$$\dot{x}_2(t) = x_1(t) \quad (80)$$

$$y(t) = x_2(t) \quad (81)$$

Solution 3.11. Consider the states $\dot{x}_2(t) = x_1(t)$ and $x_1(t) = y(t)$ With these states, the differential equation can be written

$$\begin{aligned} \ddot{x}_1(t) + 3\dot{x}_1(t) + x_1(t) = 2\dot{u}(t) &\implies \ddot{x}_1(t) + 3\dot{x}_1(t) + \dot{x}_2(t) = 2\dot{u}(t) \\ &\implies \dot{x}_1(t) + 3x_1(t) + x_2(t) = 2u(t) \end{aligned} \quad (82)$$

Thus a suitable state space model is

$$\dot{x}_1(t) = -3x_1(t) - x_2(t) + 2u(t) \quad (83)$$

$$\dot{x}_2(t) = x_1(t) \quad (84)$$

$$y(t) = x_1(t) \quad (85)$$